Looking at bare transport coefficients in fluctuating hydrodynamics

Hiroyoshi Nakano University of Tokyo

In collaboration with Yuki Minami (Gifu Univ.), and Keiji Saito (Kyoto Univ.)

arXiv:2502.15241 (2025)



Anomalous Transport in Low-Dimensional Fluids

Key features in low-dimensional systems

- Exhibit large fluctuations due to low dimensionality.
- \blacktriangleright Leads to divergence of transport coefficients with system size (L).



1D Nanowires (Experimental) Thermal conductivity (κ) diverges ~ $L^{1/3}$



2D Fluids (Theoretically) Shear viscosity (η) diverges ~ $\log L$







Our focus: A Different Angle

If yes, are they useful for predicting fluid flow?

Can we define system-size-independent transport coefficients in low-dimensional systems?



Core Idea: Wall Effects on Fluctuations and Transport

Focus: Fluid behavior near thermalized walls.



Thermalized Wall

(Wall particles follow canonical distribution at temperature)

Fluid particles also tend to thermalize near the walls (Hydrodynamic fluctuations (or long-time tail) are suppressed near walls?)



Investigate **fluctuating hydrodynamics** near walls to verify this concept.

Our expectation

Transport coefficients in bulk fluids

Transport coefficients near the wall











- 1. Discussion based on fluctuating hydrodynamics
- 3. Summary (some remarks)

2. Observation of microscopic particle system based on the MD simulation



Part 1. Discussion based on fluctuating hydrodynamics

Fluctuating hydrodynamics: our model

Fluctuating hydrodynamics explains anomalous transport & divergence of transport coefficients.

Our focus: two-dimensional fluids

density and momentum are conserved quantities (energy dynamics is ignored)



$$c_T := \sqrt{\left(\frac{\partial p}{\partial \rho}\right)_T} = \sqrt{C_{\text{press}}}$$

 $\langle \mathbf{\Pi}_{ab}^{\mathrm{ran}}(\mathbf{r},t) \rangle = 0,$

$$)\delta(t-t')\left[\eta_0\left(\delta_{ac}\delta_{bd}+\delta_{ad}\delta_{bc}\right)+\left(\zeta_0-\eta_0\right)\delta_{ab}\delta_{cd}\right],$$



Fluctuating hydrodynamics: our model

Fluctuating hydrodynamics explains anomalous transport & divergence of transport coefficients.

$$\begin{array}{l} \begin{array}{l} \displaystyle \frac{\partial\rho}{\partial t} = -\nabla\cdot(\rho\mathbf{v}) & p(\rho) = C_{\mathrm{press}}\rho & c_T := \sqrt{\left(\frac{\partial p}{\partial\rho}\right)_T} = \sqrt{C_{\mathrm{press}}} \\ \rho \left[\frac{\partial \mathbf{v}}{\partial t} + (\mathbf{v}\cdot\nabla)\mathbf{v}\right] = -\nabla p + \eta_0 \nabla^2 \mathbf{v} + \zeta_0 \nabla(\nabla\cdot\mathbf{v}) + \nabla \mathbf{\Pi}_R \end{array}$$

For some specific assumptions in our model:

Pressure

The pressure follows the equation of state for an ideal gas.

Low Mach number approximation

The ζ_0 term is neglected; we focus on only the η_0 term.

The sound velocity c_{τ} is sufficiently large and the fluids are treated as incompressible.

We focus on the dense liquid,



Viscosity in Fluctuating hydrodynamic equation

Fluctuating hydrodynamics explains anomalous transport & divergence of transport coefficients.

$$\frac{\partial \rho}{\partial t} = -\nabla \cdot (\rho \mathbf{v}) \qquad p(\rho) = C_{\text{press}}\rho \qquad c_T := \sqrt{\left(\frac{\partial p}{\partial \rho}\right)_T} = \sqrt{C_{\text{press}}}\rho \rho \left[\frac{\partial \mathbf{v}}{\partial t} + (\mathbf{v} \cdot \nabla)\mathbf{v}\right] = -\nabla p + \eta_0 \nabla^2 \mathbf{v} + \zeta_0 \nabla (\nabla \cdot \mathbf{v}) + \nabla \mathbf{\Pi}_R$$

Key assumption of fluctuating hydrodynamics framework

Fluctuating hydrodynamics derives the system-size-dependent "macroscopic" viscosity. (characterize the overall (macroscopic) fluid dissipation)

$$\eta = \eta_0 + \Delta \eta$$

 η_0 : system-size-independent quantity

system-size-dependent quantity -> divergence of macroscopic viscosity



11

First main result of our study





$$c_{press}\rho \qquad c_{T} := \sqrt{\left(\frac{\partial p}{\partial \rho}\right)_{T}} = \sqrt{C_{press}} + \zeta_{0}\nabla(\nabla \cdot \mathbf{v}) + \nabla \mathbf{\Pi}_{R}$$

The first main result

 η_0 governs the fluid motions near the walls, while $\eta = \eta_0 + \Delta \eta$ appears only in the bulk region.





We consider solving fluctuating hydrodynamics numerically



The sufficiently large C_{press} yields nearly incompressible fluids

We apply a common boundary condition in fluid dynamics

- The velocity field at the wall is $(v^x, v^y) = (v_0, 0)$ 1.
- The momentum density field at the wall is $(j^x, j^y) = (\rho v_0, 0)$ 2.

The fluid does not fluctuate at all at the solid walls

Simulation Setup

$$c_{p} = C_{press}\rho \qquad c_{T} := \sqrt{\left(\frac{\partial p}{\partial \rho}\right)_{T}} = \sqrt{C_{press}} + \zeta_{0}\nabla(\nabla \cdot \mathbf{v}) + \nabla \mathbf{\Pi}_{R}$$









Introduction of Local Viscosity

Velocity gradient is not spatially uniform



Local viscosity $-\langle \Pi^{xy} \rangle_{ss} = \eta(x) - \langle \Pi^{xy} \rangle_{ss}$ ∂x

This quantity at each specific position is defined by using the local velocity gradient at that point.

Viscosity depends on the spatial coordinate



The local viscosity decreases near the solid walls The input viscosity η_0 is observed near solid walls.







System-Size dependence of Local Viscosity



Analytical expression of velocity and local viscosity profile

We can calculate the theoretical expression for the noise-averaged Couette flow.



This calculation can be done using a perturbative expansion in ϵ (the nonlinear term).

 $\mathbf{v} = \mathbf{v}_{(0)} + \epsilon \mathbf{v}$

Several approximations were necessary to complete the calculation (the full details are omitted here)

$$\mathbf{v}_{(1)} + \epsilon^2 \mathbf{v}_{(2)} + \cdots$$









Analytical expression of velocity profile

$$\langle v^{y}(x) \rangle = \dot{\gamma}x - e^{2}\frac{\dot{\gamma}A}{L}\sum_{k_{x}}\frac{1}{k_{x}}\frac{\sin(2k_{x}x)}{2k_{x}}$$

$$\dot{\gamma} := 2v_{0}/L$$

$$A : \text{numerical factor depending on densite}$$

$$A = \frac{\rho_{0}k_{B}T}{4\eta_{0}^{2}}$$
 (within our approx

For comparison, we treat A as a fitting parameter because this calculated value can deviate from the true value due to the approximations made in the derivation.

Our derived equation accurately captures the functional form of the velocity profile.

$$k_x := \frac{\pi}{L}n$$

ity, temperature…

kimation)









Key insight from the theoretical expression

$$\eta(x) = \eta_0 \left(1 + e^2 \frac{2A}{L} \sum_{k_x} \frac{1}{k_x} \sin^2(k_x x) \right)$$

$$\dot{\gamma} := 2v_0/L$$

$$A : \text{numerical factor depending on densit}$$

$$A = \frac{\rho_0 k_B T}{4\eta_0^2} \qquad \text{(within our approx)}$$

This expression provides a key insight into the fluid's behavior

$$\eta(x=0) = \eta_0$$

near the solid walls

Our local viscosity expression is fundamental for describing the complete flow field.

$$k_x := \frac{\pi}{L}n$$

ty, temperature…

(imation)

 $\eta(x = L/2) \propto \eta_0 + C \log L$

in the bulk region



20

Part 2. Observation of microscopic particle system based on the MD simulation



Can we validate this boundary condition?

The fluid does not fluctuate at all at the solid walls.



We check whether the MD simulation results can be described by fluctuating hydrodynamics incorporating the no-slip boundary condition second main result: YES!!

Strategy of MD simulation







Setup of MD simulation

We perform molecular dynamics (MD) simulations.

In MD simulations, atoms are represented as particles that follow the classical Hamiltonian dynamics.

$$\frac{d\mathbf{r}_i}{dt} = \frac{\mathbf{p}_i}{m} \qquad \frac{d\mathbf{p}_i}{dt} = -\frac{\partial V}{\partial \mathbf{r}_i}$$

 \triangleright simple repulsive potential

$$V(r) = 10\delta^{\alpha} \text{ for } \delta > 0$$
$$V(r) = 0 \text{ for } \delta < 0$$



(particles only repel each other when they overlap)



Units for the MD simulation

atomic diameter σ

atomic mass \mathcal{M}

thermal velocity $v_{th} := \sqrt{k_B T/m}$

(or microscopic time $\tau = \sigma/v_{th}$)



Implementation of solid wall

Solid walls are implemented as a collection of particles.

1. Solid particles are trapped using an on-site potential.

$$V_{\text{onsite}}(\boldsymbol{q}) = V_0 \left[\sin(2\pi q_x) + \sin(2\pi q_y) \right]$$

2. Solid particles are thermalized using the Langevin thermostat.

$$\frac{d\boldsymbol{q}_{j}}{dt} = \frac{\boldsymbol{p}^{w}}{m}$$
$$\frac{d\boldsymbol{p}_{j}^{w}}{dt} = -\frac{\partial V_{\text{onsite}}(\boldsymbol{q}_{j} - v_{0}t\boldsymbol{e}_{x})}{\partial \boldsymbol{q}_{j}} - \sum_{i=1}^{N} \frac{\partial V_{\text{wf}}(|\boldsymbol{r}_{i} - \boldsymbol{q}_{j}|)}{\partial \boldsymbol{q}_{j}} - \gamma \boldsymbol{p}_{j}^{w} + \xi_{j}(t)$$

3. Fluid particles interact with solid particles.

4. The motion of the walls is simulated by moving the solid particles (and on-site potential) collectively at a velocity v_0 .

 $V_0 = 50$





MD simulation results



The local viscosity is observed in the same way as in the fluctuating hydrodynamics. The observed viscosity decreases near solid walls, which is consistent with the behavior in fluctuating hydrodynamics.





Changing microscopic properties of the walls

the effect of wall temperature



Freeze: Set the wall temperature to 0.

Thermal: Set the wall temperature to a finite value.

The microscopic properties of walls do not affect the results at the quantitative level.

the effect of solid-fluid interaction



hydrophilic: Use attractive solid-fluid interactions (LJ). hydrophobic: Use only repulsive solid-fluid interactions (WCA

This suggests the robustness of the results of the fluctuating hydrodynamics simulations.





Direct comparison between MD and FH simulation

Fluctuating hydrodynamics



Set the same system size, density, and temperature to match the units of both models.

MD(atomic system)



Use viscosity parameter η_0 as fitting parameters.

$$\rho \left[\frac{\partial \mathbf{v}}{\partial t} + (\mathbf{v} \cdot \nabla) \mathbf{v} \right] = -\nabla p + \eta_0 \nabla^2 \mathbf{v} + \nabla \mathbf{\Pi}_R$$





Direct comparison between MD and FH simulation





The fluctuating hydrodynamica with $\eta_0 = 0.025$ reproduces the local viscosity $\eta(x)$ of the MD simulation with high accuracy.

This strongly suggests that even in the atomic systems, η_0 governs the fluid motion near the walls.

The agreement between the two models is observed even at the atomic diameter scale.









Consistency check of best-fit viscosity parameter 1

To validate our estimate of viscosity η_0 , we compare the time correlation of the momentum density field in the bulk region in equilibrium.

$$C_{JJ}(t) := \frac{1}{2} \langle \boldsymbol{j}(\boldsymbol{r}, t) \cdot \boldsymbol{j}(\boldsymbol{r}, 0) \rangle_{\text{eq}} \qquad \boldsymbol{j} :=$$

The fluctuating hydrodynamics with $\eta_0 = 0.325$ reproduces the long-time tail of the MD simulation quantitatively with high accuracy.

The agreement between the two models is observed even at the atomic time scale.





Consistency check of best-fit viscosity parameter 2

As another consistency test, we perform the simulation of the Poiseuille flow.



The Poiseuille flow is realized by adding a constant force to entire fluids and imposing periodic boundary condition in the flow direction.



motion both near the walls and in the bulk region





Non-tiviality of the Agreement of Poiseuille Flow

Deterministic Hydrodynamics



The deterministic hydrodynamics with the viscosity observed in bulk region (η = 0.464) cannot reproduce the results of MD simulations.
Fluctuating hydrodynamics is necessary to describe fluids near walls (at least in low-dimensional systems).

Fluctuating Hydrodynamics





31

Description ability of atomic scale behaviors



Our results suggest that fluid description is possible at the mean-free path scale in such dense systems.







/ perform the MD simulations

The first main result

 η_0 governs the fluid motions near the walls, while $\eta = \eta_0 + \Delta \eta$ appears only in the bulk region.

Summary

The second main result

Even in the atomic systems, **the anomalous** transport does not occurs near the walls and η_0 governs the fluid motion near the walls.



Discussion: Physical Meaning of η_0



- \triangleright η_0 is independent of the system size.
- size-dependent macroscopic dissipation.

$$c_{p} = C_{press}\rho \qquad c_T := \sqrt{\left(\frac{\partial p}{\partial \rho}\right)_T} = \sqrt{C_{press}} + \zeta_0 \nabla (\nabla \cdot \mathbf{v}) + \nabla \mathbf{\Pi}_R$$

 \triangleright η_0 describes fluid motion at the **microscopic scale**, on the order of a particle diameter.

 $\blacktriangleright \eta_0$ characterizes **microscopic dissipation**, which is fundamentally different from the system-

$$\eta = \eta_0 + \Delta \eta$$

Our proposition : η_0 is the "bare" viscosity

This is the fundamental contribution to viscosity from the purely microscopic domain.





Appendix

UV cutoff length

Changing the UV cutoff length is related to coarse-graining process.







UV cutoff length



The predictions of fluctuating hydrodynamics depend on the value of the UV cutoff length $a_{\mu\nu}$.

The observed local viscosity changes by varying only the UV cutoff length a_{uv} with all other parameters including viscosity η_0 fixed at the same value.

Both parameters ($\eta_0, a_{\mu\nu}$) can be used as adjustable parameters.









In practice, the bare viscosity η_0^P is determined to satisfy

$$\eta_{\rm MD}(x) = \eta_{\rm FH}(x:\eta_0^P,a_{\rm uv})$$

The "practical" bare viscosity η_0^P depends on the UV cutoff length.

If (η_0, a_{uv}) lie on this relationship, any pair will reproduce the macroscopic phenomena well.







Best value of UV cutoff length

We investigated the best-fit bare viscosity for different UV cutoff lengths.



 $\eta_0 = 0.455$ for $a_{uv} = 2/3$

Colored: FHD Black: MD (all the same data) (with different $(\eta_0, a_{\mu\nu})$) well reproduced for any $a_{\mu\nu}$. The best value of UV cutoff length is about mean-free path length? (in our simulations, it is atomic diameter)

By carefully choosing η_0 , the MD results can be











The best value of UV cutoff length is about atomic diameter!!